

This is an *Accepted Manuscript*, which has been through the Royal Society of Chemistry peer review process and has been accepted for publication.

Accepted Manuscripts are published online shortly after acceptance, before technical editing, formatting and proof reading. Using this free service, authors can make their results available to the community, in citable form, before we publish the edited article. This Accepted Manuscript will be replaced by the edited, formatted and paginated article as soon as this is available.

You can find more information about *Accepted Manuscripts* in the **Information for Authors**.

Please note that technical editing may introduce minor changes to the text and/or graphics, which may alter content. The journal's standard <u>Terms & Conditions</u> and the <u>Ethical guidelines</u> still apply. In no event shall the Royal Society of Chemistry be held responsible for any errors or omissions in this *Accepted Manuscript* or any consequences arising from the use of any information it contains.



www.rsc.org/advances

Page 1 of 19

RSC Advances

1	Swirling flow structure in supersonic separators for
2	natural gas dehydration
3	Yan <u>Yang</u> ¹ , Chuang <u>Wen</u> ^{*, 1} , Shuli <u>Wang</u> ¹ , Yuqing <u>Feng</u> ² , Peter Witt ²
4	¹ Jiangsu Key Laboratory of Oil-Gas Storage and Transportation Technology,
5	Changzhou University, Zhonglou District, Changzhou, Jiangsu, 213016, China
6	² CSIRO Computational Informatics, Clayton, Victoria 3169, Australia
7	*Corresponding author. Tel.: +86 519 8329 0280, Fax: +86 519 8329 5530, Email:
8	wenchuang2008@gmail.com
9	Abstract: The supersonic separator is a novel compact tubular device for natural gas
10	dehydration. The separation mechanism is not well understood for the complicated
11	fluids with a delta wing located in the supersonic flows. We investigated the gas
12	swirling separation characteristics in supersonic velocities using the Reynolds stress
13	turbulence model. The results showed that the Laval nozzle designed with the cubic
14	polynomial and Foelsch's analytical methods formed an extremely stable and uniform
15	supersonic flow. The delta wing generated a strong swirling flow with the centrifugal
16	acceleration of around 10^7 m/s^2 to remove the condensed liquids from the mixture.
17	However, the supersonic flow was quite sensitive to the delta wing, which led to the
18	disturbance and non-uniformity of the gas dynamic parameters. This violent variation
19	in the supersonic flow would had a secondary action on the condensation, even
20	resulted in the re-evaporation of the condensed liquids.
21	Keywords: natural gas; supersonic separator; swirling flow; delta wing; numerical
22	simulation

RSC Advances Accepted Manuscript

1 1 Introduction

The removal of water is an important step for natural gas processing. The 2 3 supersonic separation is a novel technique for natural gas dehydration, working on the supersonic expansion and cyclonic separation [1]. Firstly, a Laval nozzle is usually 4 employed to accelerate the natural gas to supersonic velocities, resulting in a low 5 pressure and temperature for nucleation and condensation of water vapour and heavier 6 7 hydrocarbon components. Then, a swirl device is used to generate a strong swirling flow to separate and remove the condensed liquids from the gas-liquid mixture. Lastly, 8 9 the dry gas goes through a diffuser for the purpose of the pressure recovery.

As a result of the complexity of the supersonic separation technique, the studies 10 mainly focus on the flow simulation using the computational fluid dynamics (CFD) 11 technique, while just few experiments are performed. A supersonic separation system 12 was built on an offshore platform in Malaysia with a design capacity of 17×10^4 Nm³/d. 13 The comparison analysis showed a cost reduction of 24% for the offshore platform [2]. 14 A pilot plant was constructed in Calgary, Canada, to test the supersonic separator for 15 16 the removal of water vapour and heavier target components from natural gas [3]. The designed natural gas flow rate was up to 12 kg/s with an inlet pressure of 50-70 atm. 17 An indoor experiment loop was set up to test the dehydration characteristics of a 18 designed supersonic swirling separator with the moist air by Liu et al [4]. The dew 19 point depression between the inlet and outlet of the separator was analysed with the 20 various pressure loss ratios. A supersonic separator was compared to a Joule-Thomson 21 22 valve with TEG in an onshore plant and demonstrated the high economic performance 23 and natural gas liquids recovery of a supersonic separator [5].

Recently, some numerical studies are performed to predict the gas flows in supersonic separator or Laval nozzle for natural gas dehydration. Without considering

1 the swirling flow, Jassim et al [6, 7] employed the CFD technique to evaluate the real gas effect on the natural gas flow through a supersonic nozzle at high pressure. The 2 methane and nitrogen were adopted to analyse the changes of the shock position 3 within the real gas and ideal gas models. A constant tube was inserted between the 4 nozzle and the diffuser to study the influence of the geometry on the shock positions. 5 Karimi and Abdi [8] combined the MATLAB and HYSYS packages to predict the 6 effect of the operating parameters on natural gas flow in the high pressure Laval 7 nozzle. Malyshkina [9] obtained the distribution of gas dynamic parameters of natural 8 gas through a supersonic separator with a computational method. In his another study, 9 10 a procedure was developed to predict the separation capability of water vapor and 11 higher hydrocarbons from natural gas in a supersonic separator [10]. Jiang et al. [11] 12 employed the corrected Internally Consistent Classical Theory and Gyarmathy theory to modelling the nucleation and droplet growth of natural gas in the supersonic 13 separation process. The generalized radial basis function artificial neural networks are 14 used to optimize the geometry of a supersonic separator [12]. Rajaee Shooshtari and 15 Shahsavand developed a new theoretical approach based on mass transfer rates to 16 calculate the liquid droplet growth in supersonic conditions for binary mixtures [13]. 17 In our preliminary studies, a central body was incorporated in a supersonic separator 18 19 with a swirling device composed of vanes and an ellipsoid [14]. The effects of swirls 20 on natural gas flow in supersonic separators were computationally simulated with the 21 Reynolds stress model [15]. The particle separation characteristic in a supersonic separator was calculated using the discrete particle method [16]. Also, the real gas 22 effects and pressure recovery characteristics were simulated without considering a 23 swirling flow [17, 18]. 24

The purpose of this study is to reveal another complicated swirling flow by locating a delta wing in the downstream of the nozzle outlet. Unlike above mentioned work, it is expected to be a more difficult and challenging study since the delta wing is inserted in a supersonic flow condition. The gas parameters distribution along the axis and the radius will be obtained with Reynolds stress turbulence model, especially around the delta wing area.

7 **2 Supersonic separators**

In a supersonic separator, the delta wing can be located in a supersonic channel 8 9 behind a nozzle exit, as shown in Fig 1. Even a tiny disturbance in the supersonic 10 flows in the upstream of a delta wing will cause violent changes of the flow behavior in the downstream of the wing. Thus the Laval nozzle should be designed specifically 11 to maintain the stability of the supersonic flows. For this purpose, the cubic 12 13 polynomial, shown in Eq. (1), was employed to calculate the converging contour of the Laval nozzle. This design of the convering part will accelerate the gas flow 14 uniformly to achieve the sound speed in the throat area. Foelsch's analytical 15 16 calculation was used to design the diverging part to generate the stable supersonic flows [19]. 17

18

$$\begin{cases} \frac{D - D_{cr}}{D_1 - D_{cr}} = 1 - \frac{1}{X_m^2} \left(\frac{x}{L}\right)^3 & \left(\frac{x}{L} \le X_m\right) \\ \frac{D - D_{cr}}{D_1 - D_{cr}} = \frac{1}{\left(1 - X_m\right)^2} \left(1 - \frac{x}{L}\right)^3 & \left(\frac{x}{L} > X_m\right) \end{cases}$$
(1)

where D_1 , D_{cr} and L are the inlet diameter, the throat diameter and the convergent length, respectively. X_m =0.45. x is the distance between arbitrary cross section and the inlet, and D is the convergent diameter at arbitrary cross section of x.

4

The dimensions of the designed supersonic separator are shown in Table 1. The critical cross-section area at nozzle throat is 0.0001208 mm². The nozzle inlet and outlet areas are 0.005026 m² and 0.0002217 m², respectively.



9

10 **3 Mathematical model**

11 **3.1 Governing equations**

The natural gas can be accelerated to supersonic velocities with a Laval nozzle in a supersonic separator, and accordingly the low pressure and temperature conditions are achieved for water vapour condensation. The fluid structure of natural gas flows

Ū.

1 can be described by the conservation equations of mass, momentum and energy,

2 described as Eqs. (2) - (4).

3
$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_i} (\rho u_i) = 0$$
 (2)

4
$$\frac{\partial}{\partial t}(\rho u_i) + \frac{\partial}{\partial x_j}(\rho u_i u_j + p\delta_{ij} - \tau_{ji}) = 0$$
(3)

5
$$\frac{\partial}{\partial t}(\rho E) + \frac{\partial}{\partial x_j}(\rho u_j E + u_j p + q_j - u_i \tau_{ij}) = 0$$
(4)

6 where ρ , u, p are the gas density, velocity, and pressure, respectively. τ_{ij} is the viscous 7 stress; δ_{ij} is the Kronecker delta; E is the total energy; q_j is the heat flux; t is the time.

8 3.2 Real gas equation of state

An equation of state is necessary to calculate the physical property of fluids in 9 supersonic flows. In this simulation, the Redlich-Kwong [20] real gas equation of 10 11 state model was employed to predict gas dynamic parameters. The advantage of this equation is that it is easy to use and is often accurately represent the relation between 12 temperature, pressure, and phase compositions in binary and multicomponent systems. 13 The RK EOS only requires the critical properties and acentric factor for the 14 generalized parameters. Little computer resources are required and those lead to good 15 phase equilibrium correlation. The parameters related to calculate the gas properties 16 17 can be found in [21], including the specific heat, enthalpy, entropy, and velocity of 18 sound as a function of temperature and pressure.

19 **3.3 Turbulence model**

The Reynolds stress model (RSM) presents the characteristics of anisotropic turbulence and requires the solution of transport equations for each of the Reynolds stress components as well as for dissipation transport [22]. A strong swirling flow

presents in a supersonic separator and accordingly the RSM turbulence model is
 employed to model this fluid structure here.

3 3.4 Computational method and validation

In this case, we adopted the finite volume method to solve the governing 4 equations, while the SIMPLE algorithm [23] was employed to couple the velocity 5 field and pressure. The ANSYS FLUENT software was employed for our simulation. 6 A structured grid was generated for the Laval nozzle, the cyclonic separation section 7 and the diffuser, while the delta wing section was meshed using a tetrahedral grid due 8 9 to its complexity. The pressure boundary conditions were assigned for the inlet and outlet of the supersonic separator. No-slip and adiabatic boundary conditions were 10 specified for the walls. 11

The convergence criterion was 10^{-6} for the energy equation and 10^{-3} for all other equations. When the residuals dropped below 1×10^{-6} for the energy equation and 1×10^{-3} for all other equations with reaching stationary, while simultaneously total mass error in inlet/outlet mass flow rates was below 1×10^{-4} , the solutions were considered to reach the convergence.

In our previous studies [24], the computational fluid dynamics technology was employed to predict the effect of swirls on supersonic flows with a swirl device located in the entrance of a nozzle. The numerical results were compared with the experimental data. The agreement with each other validated that the Reynolds stress model with pressure boundary conditions could accurately predict the strong swirling flow in supersonic velocities. The details of the validation and verification can be found in [24].

RSC Advances Accepted Manuscript

1 4 Results and discussion

2	The flow characteristics of a natural gas were numerically simulated in our new
3	designed supersonic separator using our developed mathematical methods. The multi-
4	components gas mixture in Baimiao gas well of Zhongyuan Oil Field was selected for
5	the calculation. The gas composition in mole fraction is as follows: 91.36% CH ₄ ,
6	$3.63\%\ C_2H_6,\ 1.44\%\ C_3H_8,\ 0.26\%\ i-C_4H_{10},\ 0.46\%\ n-C_4H_{10},\ 0.17\%\ i-C_5H_{12},\ 0.16\%\ n-C_5H_{12},\ 0.16\%\ n-C_5H_{12$
7	C ₅ H ₁₂ , 0.03% H ₂ O, 0.45% CO ₂ , 2.04% N ₂ .

8 4.1 Gas dynamic parameters in supersonic separators

Figure 2 presents the gas dynamic parameters along the flow direction, namely, 9 the gas Mach number, the static pressure, the static temperature and the tangential 10 velocity. The gas velocity increases in the convergent part of the Laval nozzle and 11 reaches the sonic velocity at the throat. After the critical condition is achieved, the 12 13 supersonic velocity is obtained in the nozzle divergent part when the back pressure is assigned to 57% of the inlet pressure in this calculation. The gas Mach number is 14 about 2.00 at the nozzle exit. The static pressure and temperature decline in Laval 15 nozzle due to the gas contraction in converging part and the expansion in diverging 16 part, respectively. The pressure and temperature at the nozzle outlet are about 10 bar 17 and -79 °C, respectively, which creates essential conditions for the nucleation and 18 19 condensation of the water vapor and higher hydrocarbons.

When the natural gas flows run out the nozzle exit, the delta wing, located in the downstream of the nozzle exit, generates a swirling motion. The strong swirls have been obtained since the change of the velocity occurs under the conditions of the supersonic velocity. In our separator, the maximum tangential velocity is up to 300 m/s, which corresponds to a centrifugal acceleration of about 10^7 m/s². The

8

centrifugal force will swing the condensed liquid droplets onto the walls and create a liquid film. However, the gas flow is quite sensitive to the delta wing in the supersonic velocities. Once the supersonic fluid flows past the front of the delta wing, a great disturbance occurs. This disturbance causes the non-uniform distribution of the flow fields, especially the increases of the static pressure and temperature.



Figure 2 Gas dynamic parameters in a supersonic separator with a delta wing

4.2 Gas Mach number at downstream of nozzle outlet

The gas Mach number at the axial and radial direction is detailed depicted at the downstream of the Laval nozzle, as shown in Figure 3. The gas Mach number presents an extremely uniform distribution at the nozzle outlet and a certain distance downstream of it. This demonstrates that the cubic polynomial and Foelsch's analytical method are a good choice to calculate the converging and diverging curve of a Laval nozzle to generate a stable and uniform supersonic flow. However, the

RSC Advances Accepted Manuscript

delta wing located in the constant tube reduces the effective flow area, which leads to the insufficient expansion of the natural gas in this area. The radial gradient of gas Mach number near the wall increases along the axial direction, especially at the upstream of the delta wing. Also, this re-compression process will result in the increase of the gas Mach number and temperature. In this condition, the condensed droplets will re-evaporate to gas phase and decline the separation performance.



Figure 3 Gas Mach number at the downstream of the nozzle outlet

4.3 Delta wing effect

The effect of a delta wing on the gas velocities in supersonic conditions is numerically simulated and the results are shown in Figure 4. We can see that the delta wing located in the supersonic area leads to the variation of the gas Mach number at the cross section. The gas Mach number can reach a peak of about 2.00 at one edge of the delta wing, while it also appears a very small value on the other edge, i.e. 1.15. Moreover, the delta wing can generate a large tangential velocity. The tangential velocity presents an extremely non-uniform distribution at the cross section. At the tail

end of the delta wing, the tangential velocity achieves a maximum of more than 300 m/s. This indicates that the centrifugal acceleration in this area can reach 10^7 m/s², which provides a strong force to separate the condensed liquids from the mixtures.



Figure 4 Gas Mach number and tangential velocity around a delta wing

4.4 Gas swirling characteristics in cyclonic separation section

Figure 5 displays the gas Mach number, tangential velocity contours and the local velocity vector profiles in the cyclonic separation section. In the downstream of the delta wing, the gas Mach number is extremely non-uniform along the axis from 1.12 to 2.02. The gas Mach number declines along the axis in general, which indicates that the oscillation appearing in the supersonic flow will have a secondary action on the condensation, even cause the re-evaporation of the condensed liquids. The tangential

velocity also changes violently from 30 m/s to 230 m/s. It also demonstrates that the supersonic flow is quite sensitive to the delta wing. Another interesting finding is that the reverse flow does not emerge easily in the delta wing area, although the dramatic change of the gas flow field is observed behind the delta wing. This is unlike the expected phenomenon in the initial design that the delta wing installed in the supersonic zones will result in a reverse flow.



(a) Mach number (Dimensionless)



(b) Tangential velocity (m/s)



(c) Velocity vector

Figure 5 Gas dynamic parameters in the cyclonic separation section

To illustrate the non-uniformity of the gas flows in the cyclonic separation section, the gas Mach number, tangential velocity and vector profiles at the cross section are given in Figure 6. It clearly shows that the delta wing located in the supersonic velocity channel results in the turbulence of the supersonic flow. The peak of this turbulence appears just behind the end of the delta wing and then declines

slowly. It also can be seen that the center of the vortex diverges from the center of the flow channel. It reveals that it is quite complicated to generate the swirling flow at the supersonic flow regions. The delta wing used here increases the complexity and non-uniform distribution of the flow field.

However, one advantage of the current design is that the tangential velocity maintains a larger value in the whole area, which creates a strong centrifugal field for liquid droplets. The centrifugal acceleration can reach 10^{6} - 10^{7} m/s² in the cyclonic separation section. The huge rate of acceleration will generate a strong helical motion to remove the water and higher hydrocarbons.



Figure 6 Gas dynamic parameters at the cross section of the cyclonic separation

5 Conclusions

The flow structure of natural gas in a supersonic separator was simulated using the Reynolds stress turbulence model and Redlich–Kwong real gas model. A delta wing was installed in the supersonic constant tube to generate a swirling flow. The gas dynamic parameters were obtained both in the axial and radial directions. The effect of the delta wing on the flow distribution are analysed in the supersonic conditions. The centrifugal acceleration can reach 107 m/s^2 , which provides a strong force to separate the condensed liquids from the mixtures. The delta wing also causes a great disturbance to the supersonic flow and a non-uniform distribution of the gas dynamic parameter. It also demonstrates that the supersonic flow is quite sensitive to the delta wing.

Acknowledgements

This work was supported in part by the National Natural Science Foundation of China (No. 51176015, 51444005), Jiangsu Key Laboratory of Oil–Gas Storage and Transportation Technology (No. SCZ1211200004/001) and the Scientific Research Foundation of Changzhou University (No. ZMF13020057).

Nomenclature

а	[-]	constant for attractive potential of molecules
b	[-]	constant for volume
$C_{l\varepsilon}$	[-]	constant
$C_{\epsilon 2}$	[-]	constant
C_{μ}	[-]	constant
D_1	[m]	inlet diameter
D _{cr}	[m]	throat diameter
D_H	[m]	hydraulic diameter
D _{L, ij}	[-]	molecular diffusion
F _{ij}	[-]	production by system rotation
G_{ij}	[-]	buoyancy production

Ε	[J]	total energy
Ι	[-]	turbulence intensity
k	$[m^2s^{-2}]$	turbulent kinetic energy
l	[m]	turbulence length scale
L	[m]	convergent length
M_t	[-]	turbulent Mach number
p	[Pa]	static pressure
p_c	[Pa]	critical pressure
P_{ij}	[-]	stress production
q_j	[Wm ⁻²]	heat flux
R	[JK ⁻¹ mol	⁻¹] gas constant
Re	[-]	Reynolds number
Sm	[-]	source term
Sk	[-]	source term
$S_{arepsilon}$	[-]	source term
t	[s]	time
Т	[K]	temperature
T_c	[K]	critical temperature
и	[ms ⁻¹]	gas velocity
\overline{u}	[ms ⁻¹]	mean velocity
u'	[ms ⁻¹]	fluctuating velocity
V	[m ³]	gas volume
V_m	[m ³ mol ⁻¹] gas molar volume
X_m	[-]	relative coordinate

Greek letters

- δ_{ij} [-] Kronecker delta
- ε [-] turbulent dissipation rate
- ε_{ij} [-] turbulent dissipation
- μ [m²s⁻¹] gas viscosity
- μ_t [m²s⁻¹] turbulent viscosity
- ρ [kgm⁻³] gas density
- σ_k [-] turbulent Prandtl number
- σ_{ε} [-] turbulent Prandtl number
- τ_{ij} [Nm⁻²] viscous stress;
- ϕ_{ij} [-] pressure strain

References

- [1] D. Okimoto and J.Brouwer, World Oil, 2002, 223, 89-91.
- [2] M. Betting and H. Epsom, World oil, 2007, 4, 197-200.
- [3] V. I. Alferov, L. A. Baguiro, L. Dmitriev, V. Feygin, S. Imaev and J. R. Lace, *Oil Gas J.*, 2005, **103**, 53-58.
- [4] H. Liu, Z. Liu, Y. Feng, K. Gu and T. Yan, Chin. J. Chem. Eng., 2005, 1, 9-12.
- [5] P. B. Machado, J. G. M. Monteiro, J. L. Medeiros, H. D. Epsom, and O. Q. F. Araujo, *J. Nat. Gas Sci. Eng.*, 2012, 6, 43-49.
- [6] E. Jassim, M. A. Abdi and Y. Muzychka, Pet. Sci. Technol., 2008, 26, 1757-1772.
- [7] E. Jassim, M. A. Abdi and Y.Muzychka, Pet. Sci. Technol., 2008, 26, 1773-1785.
- [8] A. Karimi and M. A. Abdi, Chem. Eng. Process., 2009, 48, 560-568.
- [9] M. M. Malyshkina, High Temp., 2008, 46, 69-76.
- [10] M. M. Malyshkina, High Temp., 2010, 48, 244-250.

- [11] D. Jiang, Q. Eri, C. Wang and L. Tang, SPE projects, facilities construction 2011,6), 58-64.
- [12]B. Mahmoodzadeh Vaziri and A. Shahsavand, J. Nat. Gas Sci. Eng., 2013, 13, 30-41.
- [13] S. H. Rajaee Shooshtari and A. Shahsavand, Sep. Purif. Technol., 2013, 116, 458-470.
- [14] C. Wen, X. Cao, Y. Yang, Chem. Eng. Process., 2011, 50(7), 644-649.
- [15] C. Wen, X. Cao, Y. Yang and W. Li, *Energy*, 2012, 37, 195-200.
- [16] C. Wen, X. Cao, Y. Yang and J. Zhang, *Adv. Powder Technol.*, 2012, 23, 228-233.
- [17] C. Wen, Y. Yang, S. Wang and Y. Feng, Appl. Energy, 2014, 132, 248-253.
- [18] C. Wen, X. Cao, Y. Yang and Y. Feng, *Oil Gas Sci. Technol.*, 2014, doi: http://dx.doi.org/10.2516/ogst/2013197
- [19] K. Foelsch, J. Aero. Sci., 1949, 16, 161-166.
- [20] O. Redlichand J. N. S. Kwong, Chem. Rev., 1949, 44, 233-244.
- [21] P. Ahuja, Chemical Engineering Thermodynamics. PHI Learning Pvt. Ltd., 2009.
- [22] S. B. Pope, *Turbulent Flows*, 1st ed., Cambridge University Press, Cambridge 2000.
- [23] S. V. Patankar and D. B. Spalding, Int. J. Heat Mass Transfer, 1972, 1, 1787-1806.
- [24] C. Wen, X. Cao, Y. Yang and J. Zhang, *Chem. Eng. Technol.*, 2011, **34**, 1575-1580.



The delta wing resulted in a violent variation of the supersonic flows, inducing the

re-evaporation of the condensed liquids.